20

1

INTEGRAL SUN GEAR COUPLING

CROSS-REFERENCE TO RELATED APPLICATION(S)

This application is a divisional application of U.S. patent application Ser. No. 11/391,764 by Loc Duong, Michael E. McCune, and Louis J. Dobek, entitled "EPICYCLIC GEAR TRAIN INTEGRAL SUN GEAR COUPLING DESIGN," filed on Mar. 22, 2006 now U.S. Pat. No. 7,591,754, which is 10 hereby incorporated by reference. Further, U.S. patent application Ser. No. 12/536,650 by Loc Duong, Michael E. McCune, and Louis J. Dobek, entitled "METHOD OF MAK-ING INTEGRAL SUN GEAR COUPLING," filed on even date with this application, is a divisional application of U.S. 15 patent application Ser. No. 11/391,764 by Loc Duong, Michael E. McCune, and Louis J. Dobek, entitled "EPICY-CLIC GEAR TRAIN INTEGRAL SUN GEAR COUPLING DESIGN," filed on Mar. 22, 2006.

BACKGROUND

This invention relates to planetary gear trains. More particularly, the invention relates to a coupling system for flexibly connecting a sun gear to a rotating shaft so that the 25 reliability and durability of the gear system components are improved. The invention is useful in aircraft engines where reliability, durability and simplicity are highly desirable.

Planetary gear trains are mechanical structures for reducing or increasing the rotational speed between two rotating 30 shafts. The compactness of planetary gear trains makes them appealing for use in aircraft engines where space is at a premium.

The forces and torque transferred through a planetary gear train place tremendous stresses on the gear train components, 35 making them susceptible to breakage and wear, even under ideal conditions. In practice, conditions are often less than ideal and place additional stresses on the gear components. For example, the longitudinal axes of a sun gear, a planet carrier, and a ring gear are ideally coaxial with the longitudi- 40 a turbine engine containing a planetary gear train. nal axis of an external shaft that rotates the sun gear. Perfect or ideal coaxial alignment, however, is rare due to numerous factors including imbalances in rotating hardware, manufacturing imperfections, and transient flexure of shafts and support frames due to aircraft maneuvers. The resulting parallel 45 and angular misalignments impose moments and forces on the gear teeth, the bearings which support the planet gears in their carrier, and the carrier itself. The imposed forces and moments accelerate gear component wear and increase the likelihood of component failure in service. Thus, accelerated 50 component wear necessitates frequent inspections and part replacements which can render the engine and aircraft uneconomical to operate.

The risk of component breakage can be reduced by making the gear train components larger and therefore stronger. 55 Increased size may also reduce wear by distributing the transmitted forces over correspondingly larger surfaces. However, increased size offsets the compactness that makes planetary gear trains appealing for use in aircraft engines, and the corresponding weight increase is similarly undesirable. The 60 use of high strength materials and wear resistant coatings can also be beneficial, but escalates the cost of the gear train and therefore reduces its desirability.

Stresses due to misalignments can also be reduced by the use of flexible couplings to connect the gear train to external devices such as rotating shafts or nonrotating supports. For example, a flexible coupling connecting a sun gear to a drive

2

shaft flexes so that the sun gear remains near its ideal orientation with respect to the mating planet gears, even though the axis of the shaft is oblique or displaced with respect to a perfectly aligned orientation. Many prior art couplings, however, contain multiple parts which require lubrication and are themselves susceptible to wear. Prior art couplings may also lack adequate rigidity and strength, with respect to torsion about a longitudinal axis, to be useful in high torque applications. Misalignment can also be accommodated by a splined connection. However the motion that occurs between the contacting spline teeth in a splined connection creates friction which is highly variable and causes the flexibility of such a connection to also be variable.

In view of these shortcomings, a simple, reliable, coupling system for connecting components of a planetary gear train to external devices while accommodating misalignment there between is sought.

SUMMARY

An integral sun gear and coupling assembly has a sun gear and a spindle that is connected at a forward end to the sun gear, and a rearward end to a shaft of a planetary gear train. The spindle has at least one undulant flexible section for accommodating misalignment between the sun gear and the shaft. The flexible section has a cylindrical ring having a diameter greater than the diameter of an adjacent section of the spindle, and is joined to the spindle by two longitudinally spaced apart diaphragms. The junctures between the diaphragms and the spindle are curved in cross section on an outer side of the diaphragms to improve flexibility and minimize stress concentrations. The inner sides of the diaphragms are a straight edge perpendicular to a central axis of the shaft which result in non-symmetric contour of the diaphragm walls.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a schematic cross-sectional side elevation view of

FIG. 2 is a cross-sectional elevation view of a coupling system for the planetary gear system of the present invention.

FIG. 3 is a cross-sectional elevation view of an undulant flexible section of the present invention.

DETAILED DESCRIPTION

FIG. 1 shows a typical turbine engine 10 known in the art, which includes as its principal components one or more compressors 12, 14, one or more turbines 16, 18 for powering compressors 12, 14, combustion chamber 20, fan 22, primary exhaust 24 and fan exhaust nozzle 26. A power train such as shafts 28, 30 extends from each turbine 16, 18 to drive the corresponding compressor 12, 14. The rotary motion of one of compressors 12, 14 is conveyed to fan 22 by way of planetary gear train 32. Planetary gear train 32 reduces the rotational speed of a compressor to a speed more suitable for the efficient operation of fan 22. The principal engine components are ideally concentric with central longitudinal axis 34.

FIG. 2 is a cross-sectional elevation view of an integral coupling system 40 for planetary gear system 32 of FIG. 1, and its relationship to engine 10. Integral coupling system 40 comprises inflexible spindle 42 and at least one undulant flexible section 44 which rotate about central longitudinal

Also illustrated are compressor drive shaft 28, planet gear 46, ring gear 48, ring gear housing 50, ring gear coupling 52,